## ON AN INVERSE PROBLEM OF BENDING

# OF A PHYSICALLY NONLINEAR INHOMOGENEOUS PLATE 

## I. Yu. Tsvelodub

An elastic plate with a physically nonlinear inclusion of an arbitrary shape is considered. This plate is subjected to pure bending under the action of transverse forces and bending moments applied at the external boundary of the plate. There are no loads distributed over the surface. The problem of finding external actions that provide a necessary uniform moment state in the inclusion, i.e., prescribed constant moments and curvatures, is formulated and solved.

Key words: pure bending of an inhomogeneous plate, physically nonlinear inclusion of an arbitrary form, uniform field of moments.

An inverse problem for an elastic region with a physically nonlinear inclusion (PNI) under conditions of plane deformation or in a generalized plane stress state was considered in [1]. The task was to obtain a homogeneous stress-strain state in the PNI by choosing appropriate external forces. The problem is extended in the present paper to the case of pure bending of an elastic plate with an arbitrarily shaped PNI. The solution is constructed in a closed form.

1. Formulation of the Problem. We consider a plate of constant thickness $h$ whose mid-surface is the region $S=S_{1} \cup S_{2}$ in the plane $O x_{1} x_{2} ; S_{1}$ and $S_{2}$ are a physically nonlinear singly connected region and an isotropic linearly elastic doubly connected region with external boundaries $L_{1}$ and $L_{2}$ ( $L_{1}$ separates the regions $S_{2}$ and $S_{1}$ ). External actions (transverse forces and bending moments) are applied only at the boundary $L_{2}$, and there are no surface loads. Thus, the plate is under conditions of pure bending, and its strains $\varepsilon_{k l}$ are described by the known relations [2]

$$
\begin{gather*}
\varepsilon_{k l}\left(x_{1}, x_{2}, x_{3}\right)=-x_{3} w_{, k l}, \quad k, l=1,2, \\
w=w\left(x_{1}, x_{2}\right), \quad\left(x_{1}, x_{2}\right) \in S, \quad\left|x_{3}\right| \leq h / 2 \tag{1.1}
\end{gather*}
$$

(the subscripts after the comma indicate the derivatives with respect to the corresponding coordinate). The equilibrium equations take the form

$$
\begin{gather*}
Q_{k}=M_{k l, l}, \quad Q_{k, k}=0, \\
Q_{k}=\int_{-h / 2}^{h / 2} \sigma_{3 k} d x_{3}, \quad M_{k l}=\int_{-h / 2}^{h / 2} \sigma_{k l} x_{3} d x_{3}, \quad k, l=1,2 . \tag{1.2}
\end{gather*}
$$

In formulas (1.1) and (1.2), w is the deflection, $Q_{k}$ and $M_{k l}$ are the shear forces and moments, respectively, and $\sigma_{k l}$ are the stress components; summation is performed over repeated indices from 1 to 2 . The coordinate system $O x_{1} x_{2}$ is chosen so that $(0,0) \in S_{1}$.

The conditions of continuity of the deflection $w$, angles of mid-surface turning (i.e., normal derivatives $\partial w / \partial n$ ), bending moments $G$, and transverse forces $Q+\partial H / \partial s$ are satisfied at the boundary $L_{1}$ between the

[^0]regions $S_{1}$ and $S_{2}$. Here $G=M_{k l} n_{k} n_{l}, Q=Q_{k} n_{k}, H=M_{k l} n_{k} t_{l}$ is the torque, $n_{k}$ and $t_{k}$ are the components of the unit vectors of the normal and the tangent to the contour $L_{1}$, respectively, and $s$ is the length of the contour arc [3].

The following equalities are valid in the elastic region $S_{2}[3]$ :

$$
\begin{gather*}
M_{k l}=-D\left[(1-\nu) w_{, k l}+\nu w_{, n n} \delta_{k l}\right], \quad Q_{k}=-D w_{, n n k}, \quad k, l=1,2 \\
D=E h^{3} /\left[12\left(1-\nu^{2}\right)\right] \tag{1.3}
\end{gather*}
$$

Here $\delta_{k l}$ are the components of the plane unit tensor, $D$ is the cylindrical rigidity, $E$ is Young's modulus, and $\nu$ is Poisson's ratio.

The constitutive equations for the physically nonlinear inclusion $S_{1}$ have the form [2]

$$
\begin{gather*}
\sigma_{k l}=F_{k l}\left(\varepsilon_{m n}\right), \quad \varepsilon_{k l}=x_{3} \varkappa_{k l}, \quad \varkappa_{k l}=-w_{, k l}, \\
F_{k l}\left(-\varepsilon_{m n}\right)=-F_{k l}\left(\varepsilon_{m n}\right), \quad k, l, m, n=1,2, \tag{1.4}
\end{gather*}
$$

where $F_{k l}$ are the nonlinear differentiable functions satisfying (at $\xi_{k l} \xi_{k l} \neq 0$ ) the inequality $\partial F_{k l} / \partial \varepsilon_{m n} \xi_{k l} \xi_{m n}>0$, which is equivalent to the condition of stability of the PNI deformation process [1, 2]:

$$
\begin{equation*}
\Delta \sigma_{k l} \Delta \varepsilon_{k l}>0 \quad \text { for } \quad \Delta \varepsilon_{k l} \Delta \varepsilon_{k l} \neq 0 \tag{1.5}
\end{equation*}
$$

As was demonstrated in [2], inequality (1.5) ensures unequivocal solubility of the relations

$$
M_{k l}=2 \int_{0}^{h / 2} F_{k l}\left(x_{3} \varkappa_{m n}\right) x_{3} d x_{3}
$$

following from (1.2) and (1.4), with respect to the curvatures $\varkappa_{k l}$.
In addition, the uniformity of the field of moments $M_{k l}$ in $S_{1}$ ensures the uniformity of the field of curvatures $\varkappa_{k l}$ in $S_{1}$ [2].

An example of functions (1.4) satisfying condition (1.5) can be found in [2].
Let us formulate the main problem: Which transverse forces and bending moments have to be applied to the external boundary $L_{2}$ of the region $S_{2}$ to reach a necessary uniform moment state in the PNI, i.e., for $M_{k l}$ in $S_{1}$ to be independent of $x_{k}(k, l=1,2)$ ?
2. Solution of the Problem. It follows from Eqs. (1.2) and (1.3) that the deflection $w$ in the region $S_{2}$ in the case considered with no surface loads satisfies the biharmonic equation $w_{, k k l l}=0$; therefore, the following known presentation holds [4]:

$$
\begin{equation*}
2 w=\bar{z} \varphi(z)+z \overline{\varphi(z)}+\chi(z)+\overline{\chi(z)}, \quad z=x_{1}+i x_{2} \tag{2.1}
\end{equation*}
$$

According to (2.1), the deflection $w$ can be considered as a function of two independent complex variables $z$ and $\bar{z}$. Then, we obtain the following relations for the moments $M_{k l}$ in $S_{2}$ [3-5]:

$$
\begin{gather*}
M_{11}+M_{22}=-2 D(1+\nu)[\Phi(z)+\overline{\Phi(z)}]=-4 D(1+\nu) w_{, z \bar{z}} \\
M_{22}-M_{11}+2 i M_{12}=2 D(1-\nu)[\bar{z} \Phi \prime(z)+\Psi(z)]=4 D(1-\nu) w_{, z z}  \tag{2.2}\\
\Phi(z)=\varphi^{\prime}(z), \quad \Psi(z)=\chi^{\prime \prime}(z)
\end{gather*}
$$

In the region $S_{1}$, we have $w_{, k l}=-\varkappa_{k l}$, and $w_{, k l}$ are independent of $x_{1}$ and $x_{2}$. With accuracy to a function linear with respect to $x_{1}$ and $x_{2}$, we can find

$$
2 w=-\varkappa_{k l} x_{k} x_{l}
$$

Passing to the variables $z=x_{1}+i x_{2}$ and $\bar{z}=x_{1}-i x_{2}$, we obtain

$$
\begin{equation*}
8 w(z, \bar{z})=\left(\varkappa_{22}-\varkappa_{11}+2 i \varkappa_{12}\right) z^{2}+\left(\varkappa_{22}-\varkappa_{11}-2 i \varkappa_{12}\right) \bar{z}^{2}-2\left(\varkappa_{11}+\varkappa_{22}\right) z \bar{z} . \tag{2.3}
\end{equation*}
$$

The boundary conditions on $L_{1}$ for the functions $\Phi(z)$ and $\Psi(z)$ from (2.2), which determine the stress-stain state in $S_{2}$, have the form $[2,3]$

$$
\Phi(\tau)-\lambda_{k} \overline{\Phi(\tau)}-\left[\bar{\tau} \Phi^{\prime}(\tau)+\Psi(\tau)\right] \mathrm{e}^{2 i \alpha}=f_{k}, \quad k=1,2
$$

$$
\begin{align*}
\lambda_{1}=-1, \quad \lambda_{2} & =\frac{3+\nu}{1-\nu}, \quad f_{1}=\mathrm{e}^{i \alpha} \frac{d\left(w_{, 2}+i w_{, 1}\right)}{d s}=2 i \mathrm{e}^{i \alpha} \frac{d w_{, z}}{d s}  \tag{2.4}\\
f_{2} & =\frac{1}{D(1-\nu)}\left[G-i\left(H+\int_{0}^{s} Q d s\right)\right]
\end{align*}
$$

where $\alpha$ is the angle between the normal to the contour $L_{1}$ at the point $\tau$ and the axis $O x_{1}$. The quantities $w_{, k}$ $(k=1,2), w_{, z}, G, H$, and $Q$ in (2.4) are assumed to be defined on $L_{1}$ as functions of the arc coordinate $s$. By virtue of the above-indicated conditions of continuity, these functions can be determined by approaching $L_{1}$ from the region $S_{1}$, where the stress-strain state is prescribed, i.e., $M_{k l}$ and $\varkappa_{k l}$ are known, and the deflection $w$ is found from (2.3).

As was shown in [2], the following equalities are valid for the quantities $f_{1}$ and $f_{2}$ from (2.4):

$$
\begin{gather*}
f_{1}=-\left[\varkappa_{11}+\varkappa_{22}+\left(\varkappa_{22}-\varkappa_{11}+2 i \varkappa_{12}\right) \mathrm{e}^{2 i \alpha}\right] / 2 \\
f_{2}=\left[M_{11}+M_{22}-\left(M_{22}-M_{11}+2 i M_{12}\right) \mathrm{e}^{2 i \alpha}\right] /[2 D(1-\nu)] \tag{2.5}
\end{gather*}
$$

Let the conformal mapping of an infinite region located outside the boundary $L_{1}$ and including the region $S_{2}$ onto the exterior of the unit circumference $\gamma_{1}$ of the complex plane $\zeta$ have the form

$$
\begin{equation*}
z=\omega(\zeta)=m_{1} \zeta+\sum_{k=1}^{\infty} m_{-k} \zeta^{-k}, \quad \zeta=\rho \mathrm{e}^{i \theta} \tag{2.6}
\end{equation*}
$$

Then, from Eqs. (2.4)-(2.6) with $\rho=1$, we obtain the boundary conditions for the functions $\Phi_{1}(\zeta)=\Phi(\omega)$ ) and $\Psi_{1}(\zeta)=\Psi(\omega(\zeta))$ determining the stress-strain state for $|\zeta|>1[4,5]$ :

$$
\begin{gather*}
\Phi_{1}(\sigma)-\lambda_{k} \overline{\Phi_{1}(\sigma)}-\left[\overline{\omega(\sigma)} \Phi_{1}^{\prime}(\sigma) / \omega^{\prime}(\sigma)+\Psi_{1}(\sigma)\right] \mathrm{e}^{2 i \alpha}=\overline{F_{k}(\sigma)} \quad \text { on } \quad \gamma_{1}, \\
\overline{F_{k}(\sigma)}=f_{k} \quad(k=1,2), \quad \sigma=\mathrm{e}^{i \theta}, \quad \mathrm{e}^{2 i \alpha}=\sigma^{2} \omega^{\prime}(\sigma) / \overline{\omega^{\prime}(\sigma)} . \tag{2.7}
\end{gather*}
$$

It follows from (2.7) that

$$
\Phi_{1}(\sigma)=(1-\nu)\left[F_{1}(\sigma)-F_{2}(\sigma)\right] / 4 \quad \text { on } \quad \gamma_{1}
$$

From here, with allowance for (2.5), we find $[4,5]$

$$
\begin{gather*}
\Phi_{1}(\zeta)=A-(B+i C) \omega_{1}^{\prime}(\zeta) / \omega^{\prime}(\zeta), \quad \omega_{1}(\zeta) \equiv \bar{\omega}\left(\zeta^{-1}\right)=\overline{\omega\left(\bar{\zeta}^{-1}\right)}, \\
\Psi_{1}(\zeta)=\left\{\left[\bar{F}_{1}\left(\zeta^{-1}\right)-\Phi_{1}(\zeta)-\bar{\Phi}_{1}\left(\zeta^{-1}\right)\right] \omega_{1}^{\prime}(\zeta)-\Phi_{1}^{\prime}(\zeta) \omega_{1}(\zeta)\right\} / \omega^{\prime}(\zeta) \\
\bar{F}_{1}\left(\zeta^{-1}\right)=A_{1}-\left(B_{1}-i C_{1}\right) \omega^{\prime}(\zeta) / \omega_{1}^{\prime}(\zeta), \quad|\zeta|>1  \tag{2.8}\\
8 A=2(1-\nu) A_{1}-\left(M_{11}+M_{22}\right) / D, \quad 8 B=2(1-\nu) B_{1}+\left(M_{22}-M_{11}\right) / D \\
4 C=(1-\nu) C_{1}-M_{12} / D, \quad 2 A_{1}=-\left(\varkappa_{11}+\varkappa_{22}\right), \quad 2 B_{1}=-\left(\varkappa_{22}-\varkappa_{11}\right), \quad C_{1}=\varkappa_{12}
\end{gather*}
$$

It follows from (2.6) and (2.8) that the functions $\Phi_{1}(\zeta)$ and $\Psi_{1}(\zeta)$ are holomorphic in the ring $1<|\zeta|<R$, where $R^{-1}=\lim \left|m_{-n}\right|^{1 / n}$ as $n \rightarrow \infty$. If the number of terms under the summation $\operatorname{sign}$ in (2.6) is finite, then $R=\infty$. If the contour $\gamma_{2}$ of the plane $\zeta$ corresponding to the boundary $L_{2}$ lies inside this ring, the sought external actions on $L_{2}$ (transverse forces and bending moments) are determined by the known formulas of the form (2.4) $[3,4]$.

It should be noted that the solution constructed in the region $S_{2}$ can be continued beyond the boundary $L_{2}$, like in [1], if the corresponding values are $|\zeta|<R$. In particular, if the contour $L_{1}$ is an ellipse, the continuation is also possible for $|\zeta| \rightarrow \infty$. This corresponds to the case of pure bending of the plate with an elliptic PNI under the action of uniformly distributed moments at infinity [2].
3. Uniqueness of the Problem Solution. It follows from dependences (2.8) that the solution for the stress-strain state in $S_{2}$ exists and is unique if the stress-strain state in $S_{1}$ is prescribed and the contour $\gamma_{2}$ corresponding to $L_{2}$ lies inside the ring $1<|\zeta|<R[1]$. The reverse statement is also valid: for prescribed bending moments $G$ and transverse forces $Q+\partial H / \partial s$ on $L_{2}$, the moments $M_{k l}$ and curvatures $\varkappa_{k l}$ in $S=S_{1} \cup S_{2}$ are
determined uniquely, i.e., the above-considered stress-strain state is formed in the region $S$ under the action of the force actions on $L_{2}$ found; hence, the PNI is in a uniform moment state.

The proof of this statement is similar to that given in [1]. In this case, we use condition (1.5) and the equation of virtual works, which has the following form in the case considered with pure bending of plates under the above-indicated conditions of continuity on $L_{1}$ [2]:

$$
\int_{-h / 2}^{h / 2} \int_{S} \sigma_{k l} \varepsilon_{k l} d S d x_{3}=\int_{L_{2}}\left[\left(Q+\frac{\partial H}{\partial s}\right) w-G \frac{\partial w}{\partial n}\right] d s
$$

This equation is valid for all fields of $\sigma_{k l}$ and $\varepsilon_{k l}$ independent of each other; thereby, $\varepsilon_{k l}$ and $w$ satisfy relations (1.1), while $Q_{k}$ and $M_{k l}$ satisfy the equilibrium equations (1.2). The same equation was used in [2] to prove the uniqueness of the solution of the problem on pure bending of an infinite plate with an elliptic PNI.

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[^0]:    Lavrent'ev Institute of Hydrodynamics, Siberian Division, Russian Academy of Sciences, Novosibirsk 630090; itsvel@hydro.nsc.ru. Translated from Prikladnaya Mekhanika i Tekhnicheskaya Fizika, Vol. 48, No. 5, pp. 104-107, September-October, 2007. Original article submitted October 6, 2006.

